

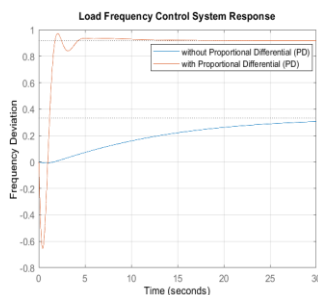
Performance Optimization of Hydraulic Type Load Frequency Control in Hydropower Plants: Study of Controllers with and Without Droop Characteristics

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Graphical Abstract



Abstract

This research discusses the performance of the hydraulic type Load Frequency Control (LFC) system in hydropower plants using a single controller configuration equipped with a governor against power input. The Load Frequency Control (LFC) system is used to stabilize the power frequency during load changes by considering the influence of droop characteristics on the governor. Analysis is performed using MATLAB simulation to measure transient parameters such as rise time, peak time, steady state time, and maximum skip. The results show that the use of droop on the governor is more effective in reducing overshoot, improving system stability, but slowing down the response time. Conversely, the configuration without droop accelerates the system response but risks overshoot. Controllers with derivative components such as PD and PDF, are superior in achieving rise time. Whereas, controllers that have integral components such as PI, PID, and PIDF are better at controlling overshoot when using droop characteristics. In conclusion, the combination of droop characteristics with derivative components in the governor provides optimal performance in maintaining the frequency stability of the hydraulic type Load Frequency Control (LFC) system.

Keywords: Load Frequency Control, Governor, Droop, PID Controller, Hydraulic



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1. INTRODUCTION

Electrical energy is a major necessity in everyday life. The stable quality of electricity supply, especially in terms of frequency, is very important to maintain the optimal performance of equipment used by consumers. The power system consists of power plants that produce energy and loads that utilize that energy[1]. When present at load, the power system experiences frequency changes that can have a negative impact if not addressed immediately. Therefore, a control system is needed that can stabilize the frequency of electricity. One of the control systems used is Load Frequency Control (LFC)[2][3].

The variation of consumer and production loads in the electricity system leads to instability of active and reactive power requirements. This causes inappropriate frequency changes during system operation[4][5][6]. Larger frequency deviations can lead to poor system operation or lower performance. This can endanger consumers and can cause blackouts if the frequency is unstable or drops too low, which can cause damage to expensive production equipment[7]. The frequency must be set at 50Hz or at a tolerance limit of $\pm 2\%$ of the normal frequency to ensure good electricity quality. So in this case a Load Frequency Control (LFC) control system is needed to prevent frequency changes[8][9].

A frequency control system or Load Frequency Control (LFC) is excellent for controlling energy supply because it serves to store frequency changes and monitor frequency instability caused by load changes. Frequency

and data flow in an area tend to change when there is a disturbance or load variation, so it must be controlled to remain stable[10]. By using Load Frequency Control (LFC), frequency fluctuations in the electrical system caused by load changes should not exceed the tolerance limit[11][12][13]. Load Frequency Control (LFC) control system is one of the most important parts in power systems and control systems. Load Frequency Control (LFC) balances the power output generated by power plants and the energy expected by consumers to provide a stable and high-quality power supply[14][15]. Studies on superconducting magnetic energy storage support Load Frequency Control (LFC) technology. These studies show that superconducting magnetic energy storage can improve overall system stability and response[4][5][14][18].

One of the methods used by Load Frequency Control (LFC) systems is the Proportional-Integral-Derivative (PID) control controller[12][19]. Due to its ease of use and outstanding performance, PID controllers are still the top choice in the power industry today[6][7][21]. Controller parameters such as proportional constant (K_p), integral constant (K_i), and derivative constant (K_d) are used to calculate the output value of the PID controller[22]. Mechanical valve action is provided by the PID controller based on the magnitude of the resulting error[23][24][25]. Often, PID controllers are used in Load Frequency Control (LFC) systems but these controllers have drawbacks in terms of dynamic work[8][16] and can cause instability if the gain values are not appropriate.

This research aims to analyze the performance of Load Frequency Control (LFC) system in hydroelectric power plant with single controller configuration using governor against power input. The use of governor in this system aims to maintain a stable frequency despite load changes and the analysis results can be seen through the performance of transient parameters such as rise time, peak time, steady state time, and maximum pass. In addition, the research aims to compare the performance of the system using a single controller with a governor against the power input with the design criteria as a reference.

This research has limitations to ensure the focus of the Load Frequency Control (LFC) system performance analysis on hydraulic type power systems using PID controllers and filters. These limitations are needed so that the research remains directed and in accordance with the objectives achieved. The problem limitations in this research are as follows:

- The research only focuses on analyzing the performance of hydraulic-type LFC systems using several types of PID controllers (P, PI, PD, PID, PDF, and PIDF).
- The study did not include other controller configurations, such as cascade or multi-control systems.
- The system response under review is limited to the power input to the hydraulic turbine with the analyses covering conditions with and without droop characteristics of the governor.
- The research only uses MATLAB simulation to evaluate the transient response of the system, such as rise time, peak time, steady state time, and maximum skip.

The analyzed system operates at a nominal frequency of 50Hz under non-transient conditions so it does not include frequency disturbances outside the standard tolerance limits.

2. METHOD

MATLAB simulation is used to analyze the performance of a single controller on a hydraulic LFC system against power input. The system block diagram was drawn to show the control flow, including details of the parameters tested, such as rise time, peak time, steady time, and maximum skip, as well as the settings for the governor and droop.

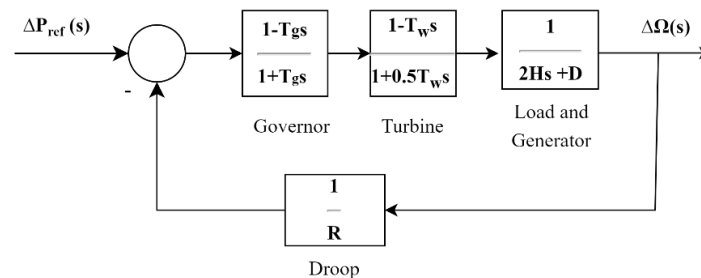


Figure 1. Block Diagram of Uncontrolled Hydraulic Type LFC against Power Input

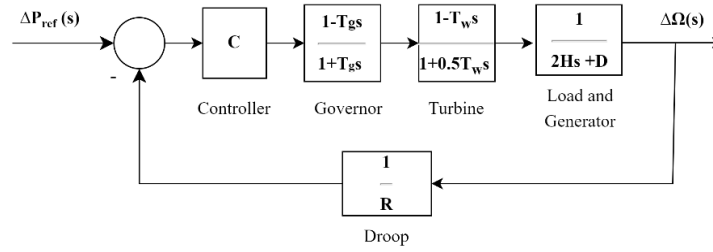


Figure 2. Block Diagram of Hydraulic Type LFC with Single Controller against Power Input

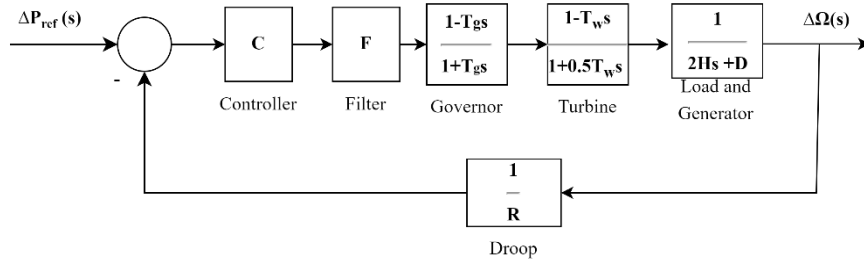


Figure 3. Block Diagram of Hydraulic Type LFC with Single Controller using Filter against Power Input

Based on Figure 1 - Figure 3, each of the block diagrams has a transfer function. The transfer function can be seen in the following equation. Figure 1 is a block diagram of a hydraulic type LFC without a controller, where the equation can be seen as follows.

$$\frac{\Delta\Omega(s)}{\Delta P_{ref}(s)} = \frac{\left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right)}{1 + \left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right) \left(\frac{1}{R}\right)} \quad (1)$$

In Figure 2. is a block diagram of a hydraulic type LFC with a controller where the equation can be seen below.

$$\frac{\Delta\Omega(s)}{\Delta P_{ref}(s)} = \frac{(C) \left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right)}{1 + (C) \left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right) \left(\frac{1}{R}\right)} \quad (2)$$

In Figure 3. is a block diagram of hydraulic type LFC with a controller using a filter, where the equation can be seen below.

$$\frac{\Delta\Omega(s)}{\Delta P_{ref}(s)} = \frac{(C)(F) \left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right)}{1 + (C)(F) \left(\frac{1}{1+T_G s}\right) \left(\frac{1-T_w s}{1+\frac{1}{2}T_w s}\right) \left(\frac{1}{2Hs+D}\right) \left(\frac{1}{R}\right)} \quad (3)$$

Table 1. Types of PIDTune Controllers Standard Model[12]

Controller Type	Mathematical Representation
Proportional (P)	K_p
Proportional-Integral (PI)	$K_p + \frac{K_p}{T_i s}$
Proportional-Differential (PD)	$K_p + K_p T_d s$
Proportional-Integral-Differential (PID)	$K_p + \frac{K_p}{T_i s} + K_p T_d s$
Proportional-Differential with a first-order filter in the Differential section (PDF)	$K_p + \frac{K_p T_d s}{N s + 1}$
Proportional-Integral-Differential with a first-order filter in the Differential section (PIDF)	$K_p + \frac{K_p}{T_i s} + \frac{K_p T_d s}{N s + 1}$

Table 2. Design Criteria for Analyzing the Switching of Hydraulic Type Load Frequency Control to Power Input

Design Criteria	Design Value
Rise Time (T_r)	<2.000 s
Peak Time (T_p)	<4.000 s
Settling Time (T_s)	<6.000 s
Peak Value (y_p)	<0.055
Maximum Pass Rate (M_p)	<20%

3. RESULTS AND DISCUSSION

In this section, it can be seen that the use of a governor in a hydraulic-type Load Frequency Control system has an important role in improving the transient performance of the system. The use of droop characteristics in the governor results in better stability as it can reduce overshoot, albeit with a little extra time at the peak time and steady state time. Without droop, the system tends to have a faster response time but risks overshoot that can affect long-term stability. The results of the analysis are obtained from the comparison of the criteria set.

Table 3. Information Analysis of Hydraulic Type Switching with Single Controller against Power Input without Droop Characteristics

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	2.160	13.075	0.597	13.648	1.699	13.646
Peak Time	6.344	30.312	6.938	31.367	5.209	31.279
Settling Time	10.742	52.022	3.882	51.926	7.817	52.080
Peak Value	0.989	1.111	0.935	1.074	1.009	1.077
Maximum Pass Rate	12.697	11.087	1.949	7.399	14.361	7.648

Based on Table 3, it can be seen that all controllers meet the maximum skip value and only PD and PDF controllers meet the rise time parameter. This shows that PD and PDF controllers have derivative and Proportional components that make the system more responsive in overcoming load changes in the Load Frequency Control (LFC) system. The derivative component in the PD and PDF controllers provides a fast reaction to changes in the initial error so as to accelerate the system response and suppress the rise time. Meanwhile, controllers that use integral components such as PI, PID, and PIDF experience longer rise times due to the presence of integral components that work in reducing long-term errors rather than initial responses. However, the absence of droop characteristics in these configurations allows all controllers to fulfil maximum pass through as the system can produce sharp and controlled responses without any additional restrictions on the initial frequency rise.

Table 4. Information Analysis of Hydraulic Type Switching with Single Controller against Power Input with Droop Characteristics

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	Nan	4.309	0.009	3.580	0.005	3.457
Peak Time	Inf	32.213	1.291	8.235	1.338	8.592
Settling Time	NaN	17.398	11.936	15.592	NaN	16.473
Peak Value	Inf	0.999	1.238	1.010	1.235	1.062
Maximum Pass Rate	NaN	0	3229.600	0.963	6420.800	6.238

Based on Table 4, it can be seen that only the PD and PDF controllers fulfil the parameters of rise time and peak time, while for maximum skip only the PI, PID, and PIDF controllers do. The existence of droop characteristics can slow down the initial response of the system to prevent excessive response that causes large overshoot. The derivative components in the PD and PDF controllers are able to overcome this slowdown resulting in appropriate rise times and peak times. On the other hand, for the maximum skip value parameter, the integral component in PI, PID, and PIDF controllers is able to work optimally with the droop characteristic. The integral component plays a role in reducing the final error and provides additional stability to the response so as to hold the overshoot within tolerable limits. Thus, the droop characteristic gives a significant advantage to controllers that have an integral component in keeping the maximum skip low, but can reduce the effectiveness of the rise time in controllers that do not have a derivative component.

Table 5. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input without Droop Characteristics ($\tau = 0,025$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	2.139	13.038	0.566	13.606	1.686	13.604
Peak Time	6.346	30.247	2.021	31.325	5.189	31.226
Settling Time	10.670	51.978	5.349	51.891	7.867	52.044
Peak Value	0.996	1.112	0.972	1.075	1.019	1.077
Maximum Pass Rate	13.482	11.158	5.965	7.452	15.469	7.703

Table 6. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input without Droop Characteristics ($\tau = 0,05$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	2.122	13.001	0.558	13.563	1.671	13.562
Peak Time	6.346	30.254	2.172	31.239	5.261	31.189
Settling Time	10.608	51.933	5.953	51.855	7.915	52.008
Peak Value	1.003	1.112	1.012	1.075	1.029	1.078
Maximum Pass Rate	14.289	11.230	10.317	7.507	16.598	7.758

Table 7. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input without Droop Characteristics ($\tau = 0,075$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	2.107	12.964	0.561	13.521	1.663	13.520
Peak Time	6.343	30.219	2.254	31.169	5.308	31.169
Settling Time	10.555	51.889	6.340	51.820	7.972	51.971
Peak Value	1.010	1.113	1.046	1.076	1.038	1.078
Maximum Pass Rate	15.111	11.303	14.018	7.562	17.712	7.814

Table 8. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input without Droop Characteristics ($\tau = 0,1$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	2.094	12.927	0.572	13.479	1.658	13.479
Peak Time	6.344	30.141	2.363	31.117	5.349	31.089
Settling Time	10.512	51.845	6.662	51.784	8.031	51.935
Peak Value	1.017	1.114	1.074	1.076	1.048	1.079
Maximum Pass Rate	15.944	11.377	17.111	7.618	18.814	7.871

Based on Table 5 - Table 8, it can be seen that all controllers fulfil the maximum pass value parameter at all τ values. At the value of $\tau = 0.025 - 0.05$, the PD controller fulfils all the parameters of both rise time, peak time, steady state time, and maximum skip value. This is because at low values of τ , the filter does not slow down the response significantly which allows the derivative component to work optimally in achieving a fast and stable response. However, at higher values of τ , the filter effect slows down the system response so that the steady state time is not achieved according to the set criteria. Whereas, at rise time only the PDF controller fulfils at all values of τ . This is because the filter component at high τ degrades the Differential effect so that the overall performance at steady state time and maximum skip is affected.

Table 9. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input with Droop Characteristics ($\tau = 0,025$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	NaN	4.285	0.009	3.558	0.005	3.449
Peak Time	Inf	31.859	1.410	8.158	1.349	8.704
Settling Time	NaN	17.450	NaN	15.657	NaN	16.564
Peak Value	Inf	0.999	1.280	1.015	1.267	1.068
Maximum Pass Rate	NaN	0	3344.900	1.457	6588.100	6.753

Table 10. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input with Droop Characteristics ($\tau = 0,05$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	NaN	4.262	0.010	3.545	0.005	3.440
Peak Time	Inf	31.495	1.489	8.324	1.498	8.808
Settling Time	NaN	17.505	NaN	15.725	NaN	16.660
Peak Value	Inf	0.999	1.294	1.019	1.288	1.072
Maximum Pass Rate	NaN	0	3380.100	1.938	6702.200	7.237

Table 11. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input with Droop Characteristics ($\tau = 0,075$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	NaN	4.243	0.011	3.529	0.006	3.430
Peak Time	Inf	31.120	1.573	8.241	1.579	8.904
Settling Time	NaN	17.564	NaN	15.796	NaN	16.757
Peak Value	Inf	0.999	1.286	1.024	1.279	1.077
Maximum Pass Rate	NaN	0	3359.300	2.420	6653.400	7.201

Table 12. Information Analysis of Hydraulic Type Switching with Single Controller and Filter against Power Input with Droop Characteristics ($\tau = 0,1$)

Transition	P	PI	PD	PID	PDF	PIDF
Rise Time	NaN	4.224	0.012	3.517	0.006	3.427
Peak Time	Inf	30.735	1.649	8.396	1.654	8.782
Settling Time	NaN	17.625	23.326	15.871	23.487	16.859
Peak Value	Inf	0.999	1.266	1.029	1.259	1.082
Maximum Pass Rate	NaN	0	3307.300	2.897	6546.500	8.174

Based on Table 9 - Table 12, it can be seen that the PD and PDF controllers fulfil the parameters of rise time and peak time. Meanwhile, the maximum skip value only PI, PID and PIDF controllers fulfil. The existence of droop characteristics and filters with various τ in this configuration is useful to withstand the initial response spike and maintain stability in the system response. The PD and PDF controllers fulfil the rise time and peak time, due to the presence of derivative components that help speed up the initial response while the droop and filter characteristics slow down the total response. In contrast, PI, PID, and PIDF controllers that have integral components are able to contain and control excessive overshoot. However, at high τ filters can extend the time it takes for the system to reach steady state so no controller is able to fulfil the steady state time criterion at all available values of τ .

4. CONCLUSION

This study shows that the use of a governor has an important role in stabilizing the frequency especially during load changes. The use of droop characteristics in the governor helps in reducing overshoot and improving stability although it lengthens the response time of the system. In contrast, a configuration without droop results in faster response, but with greater risk of overshoot.

The PD and PDF controllers proved to be optimal in accelerating the rise time. Whereas, PI, PID, and PIDF controllers control overshoot better when droop is applied. The use of filters with high τ values can slow down the time to reach steady state which affects the long-term stability. Overall, the combination of droop and derivative components in the governor provides optimal performance for Load Frequency Control (LFC) systems in hydraulic power systems.

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